monet-4

How does qualitative reasoning help in design?

- 1. Find qualitative properties of initial design.
- 2. Transform design to eliminate bad properties and add or ensure good ones.
- 3. Reach a design with the correct qualitative properties.
- 4. Construct a qualitative proof of correctness.
- 5. Accumulate the algebraic and numerical constraints required for the proof to hold.
- 6. Any remaining degrees of freedom may be used to optimize the design.

- Design classical controllers in their own operating regions.
- Overlapping operating regions defined by fuzzy set membership functions.
 - "appropriateness measures" for the controller
- The output of the heterogeneous controller is
 - the average output of the local controllers,
 - weighted by their appropriateness measures.
- Classical optimality within each region; Smooth transitions.
- Example: the water tank controller



$$\dot{x} = f(x, u) = q - u \cdot p(x).$$

- x =amount in tank
- q = inflow into tank
- u = drain area
- p(x) =influence of pressure at drain

The operating regions and their appropriateness measures:



The local control laws:

$$\begin{aligned} x \in Low \ \Rightarrow \ u_l(x) &= 0 \\ x \in Normal \ \Rightarrow \ u_n(x) &= k(x - x_s) + u_s \\ x \in High \ \Rightarrow \ u_h(x) &= u_{max} \end{aligned}$$

The global control law:

$$u(x) = l(x)u_l(x) + n(x)u_n(x) + h(x)u_h(x).$$

The discrete abstraction:

$$\underline{\text{Low}} \longrightarrow \boxed{\text{Normal}} \longleftarrow \boxed{\text{High}}.$$

• Overlapping operating regions for the local laws.



• Require qualitative agreement where laws overlap.



• Guarantee monotonic behavior in overlap regions.

$$Low \Rightarrow q > 0$$

 $Normal \Rightarrow q_b < q < q_c$
 $High \Rightarrow q < u_{max} \cdot p(c)$

• Abstract the control law to a finite transition diagram.

- Simple but general qualitative models have reliable properties.
- The tank model: $\dot{x} = -f(x)$, where $f \in M^+$.

– Converges to stable fixed point: $f(x) \longrightarrow 0$.

• The undamped spring: $\ddot{x} = -h(x)$, where $h \in M_0^+$.

- Periodic oscillation.

- The damped spring: $\ddot{x} = -h(x) f(\dot{x})$, where $f, h \in M_0^+$.
 - Converges to stable fixed point: $x \longrightarrow 0$.
 - Converges via spiral or nodal trajectory, depending on $sign(b^2 4c)$, where b = f'(0) and c = h'(0).

We can design qualitative behaviors by matching these models in local operating regions.

Design the Inverted Pendulum (applying torque at the pivot)

• The pendulum is: $\ddot{\theta} = -g\sin\theta - f(\dot{\theta})$.

-(0,0) is a stable attractor,

– $(\pi, 0)$ is an unstable saddle.

- With controller: $\ddot{\theta} = -g\sin\theta f(\dot{\theta}) + u(\theta, \dot{\theta})$.
- Design $u(\theta, \dot{\theta})$ so the model has the right qualitative behavior.
- Make (0, 0) a spiral repellor by adding a "negative friction" component $p(\dot{\theta})$.
- Make (π, 0) a stable attractor by adding a "spring force" component q(θ π).
- Constraints follow from requirement that net forces be monotonic functions of θ or $\dot{\theta}$.
- Another constraint follows from need to get from nbd(0,0) to nbd(π,0).

The cart-pole version of the problem is a little more complicated.

Near (θ, θ) = (0, 0), the pendulum is qualitatively a damped spring.

$$\ddot{ heta} = -h(heta) - f(\dot{ heta})$$

 $\ddot{ heta} = -g\sin heta - f(\dot{ heta})$

- The friction term $f(\dot{\theta})$ is responsible for the spiral in.
- We can get spiral out behavior if we have "negative friction" (pumping).
- \bullet Define the controller $u(\theta,\dot{\theta})=p(\dot{\theta})$ such that

$$p(\dot{\theta}) - f(\dot{\theta}) \in M_0^+.$$

• The result is a "negatively damped spring":

$$\ddot{\theta} = -g\sin\theta - f(\dot{\theta}) + p(\dot{\theta})$$

that will spiral away from (0, 0).

- Near $(\theta, \dot{\theta}) = (\pi, 0)$, the pendulum has a saddle point.
- Changing variables to $\phi = \theta \pi$ gives

$$\ddot{\phi} = g \sin \phi - f(\dot{\phi})$$

with "negative spring force" making $(\pi, 0)$ unstable.

• Define the controller $u(\phi,\dot{\phi})=-R(\phi)$ such that

$$R(\phi) - g\sin\phi \in M_0^+.$$

• The result is a damped spring:

$$\ddot{\phi} = -R(\phi) + g\sin\phi - f(\dot{\phi})$$

which converges $\phi \longrightarrow 0$.

Can Pumping Reach from Bottom to Top?

- Let θ_1 be the *minimum* angle from which the Top controller can pick up the stopped pendulum.
- Let θ_0 be the maximum angle reached on the *preceding* cycle.
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- In the end, the constraint is:

• The Controllers

- Bottom: $\ddot{\theta} = -g\sin\theta f(\dot{\theta}) + p(\dot{\theta})$
- Top: $\ddot{\theta} = -R(\theta \pi) + g\sin(\theta \pi) f(\dot{\theta})$
- There is also a "Fast" region, to stop spinning.

• The Constraints

- **Bottom**: $p(\dot{\theta}) f(\dot{\theta}) \in M_0^+$.
- **Top:** $R(\phi) g \sin \phi \in M_0^+$.
- Transition:
- The Regions:
 - **Bottom**: neighborhood around $(\theta, \dot{\theta}) = (0, 0)$.
 - Top: neighborhood around $(\theta, \dot{\theta}) = (\pi, 0)$.
 - Fast:

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• Benjamin J. Kuipers and Karl J. Åström. 1994. The composition and validation of heterogeneous control laws. *Automatica* **30**(2), February 1994.